

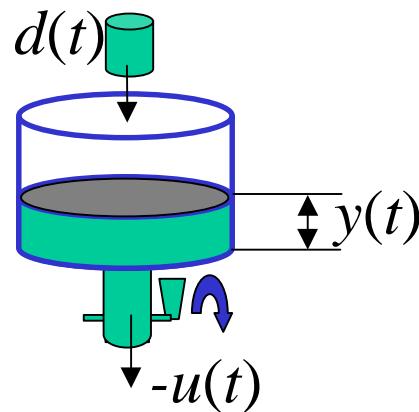
Lecture 4 - PID Control

- 90% (or more) of control loops in industry are PID
- Simple control design model → simple controller

P control

- Integrator plant:

$$\dot{y} = u + d$$



- P controller:

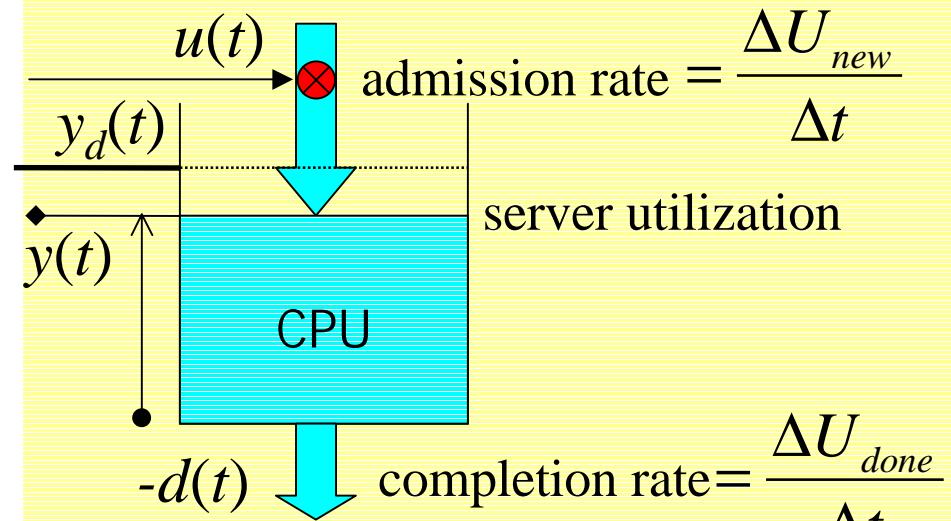
$$u = -k_p(y - y_d)$$

Example:

Utilization control in a video server

Video stream i

- processing time $c[i]$, period $p[i]$
- CPU utilization: $U[i] = c[i]/p[i]$



P control

- Closed-loop dynamics

$$\dot{y} + k_P y = k_P y_d + d$$

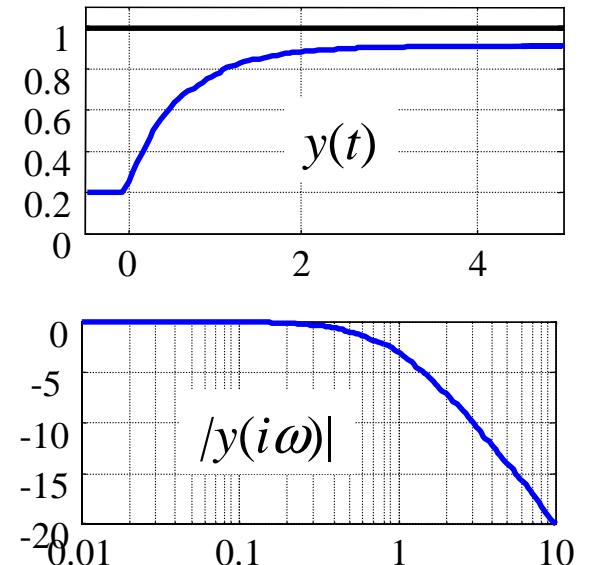
$$y = \frac{k_P}{s + k_P} y_d + \frac{1}{s + k_P} d$$

- Steady-state ($s = 0$) $y_{ss} = y_d + \frac{1}{k_P} d_{ss}$
- Transient $y(t) = y(0)e^{-t/T} + \left(y_d + \frac{1}{k_P} d_{ss} \right) \cdot \left(1 - e^{-t/T} \right)$
 $T = 1/k_P$

- Frequency-domain (bandwidth)

$$y_d(t) = \hat{y}_d(i\omega) e^{i\omega t} \quad |\hat{y}(i\omega)| = \frac{|\hat{y}_d(i\omega) + \hat{d}(i\omega)/k_P|}{\sqrt{(\omega/k_P)^2 + 1}}$$

$$d(t) = \hat{d}(i\omega) e^{i\omega t}$$



I control

$$y = g \cdot u + d,$$

- Introduce integrator into control

$$\dot{u} = v,$$

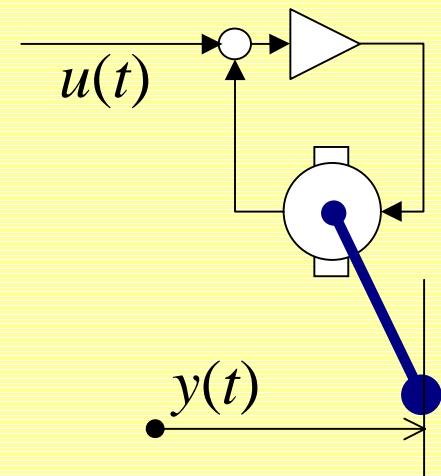
$$v = -k_I(y - y_d)$$

- Closed-loop dynamics

$$y = \frac{gk_I}{s + gk_I} y_d + \frac{s}{s + gk_I} d$$

Example:

- Servosystem command



- More:

- stepper motor
- flow through a valve
- motor torque ...

Sampled time I control

- Step to step update:

$$y(t) = g \cdot u(t) + d(t)$$

$$u(t) = u(t-1) + v(t-1)$$

$$v(t) = k_I [y(t) - y_d]$$

*sampled time
integrator*

- Closed-loop dynamics

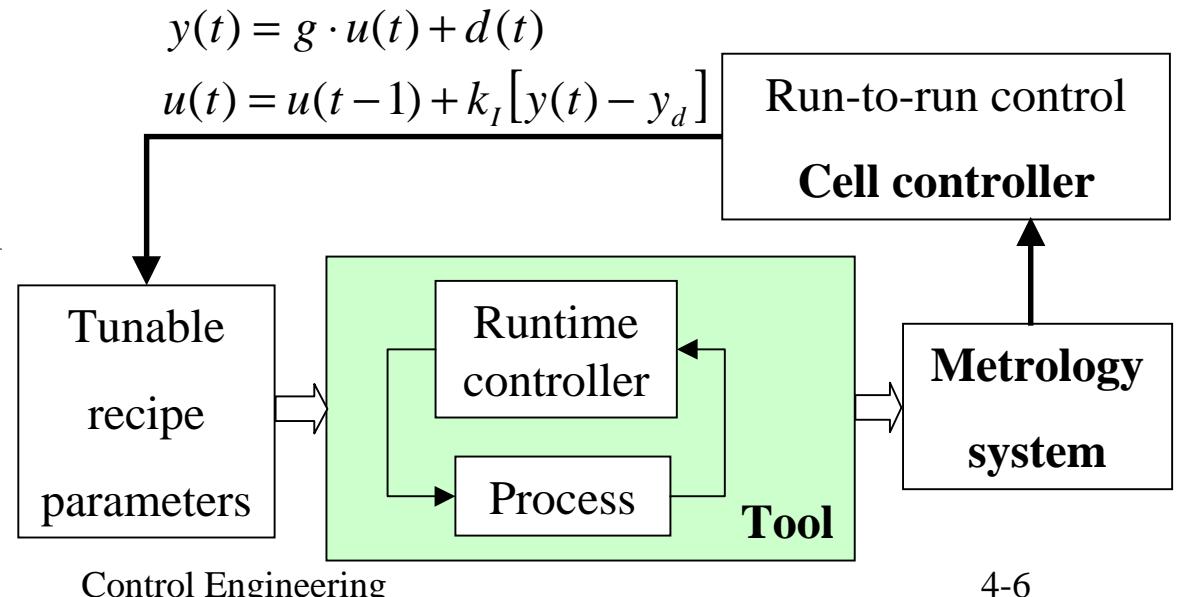
$$y = g \cdot u + d \quad \longrightarrow \quad y = \frac{gk_I}{z-1 + gk_I} y_d + \frac{z-1}{z-1 + gk_I} d$$

$$u = \frac{k_I}{z-1} [y - y_d]$$

- Deadbeat control: $gk_I = 1 \quad \longrightarrow \quad y = z^{-1} y_d + (1 - z^{-1}) d$

Run-to-run (R2R) control

- Main APC (Advanced Process Control) approach in semiconductor processes
- Modification of a product recipe between tool "runs"
- Processes:
 - vapor phase epitaxy
 - lithography
 - chemical mechanical planarization (CMP)
 - plasma etch



PI control

- First-order system:

$$\tau \dot{y} = -y + u + d$$

- P control + integrator for cancelling steady state error

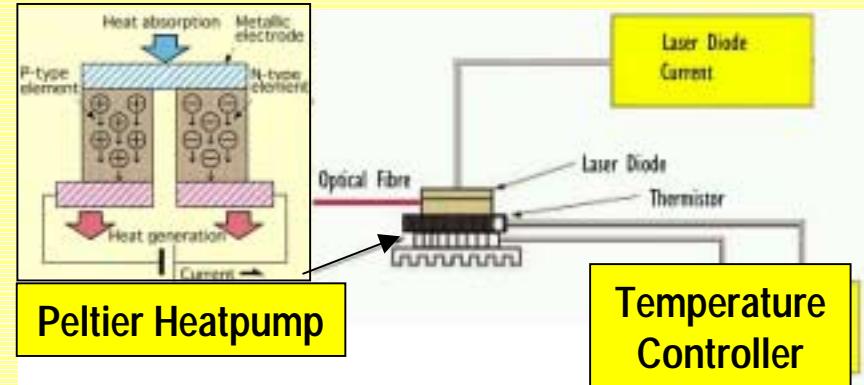
$$e = y - y_d;$$

$$\dot{v} = e$$

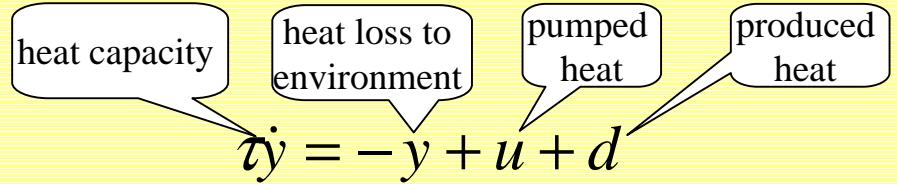
$$u = -k_I v - k_P e$$

Example:

- WDM laser-diode temperature control



$$y(t) = \text{temperature} - \text{ambient temperature}$$



- Other applications

- ATE
- EDFA optical amplifiers
- Fiber optic laser modules
- Fiber optic network equipment

PI control

- P Control + Integrator for cancelling steady state error

$$e = y - y_d;$$

$$\dot{v} = e$$

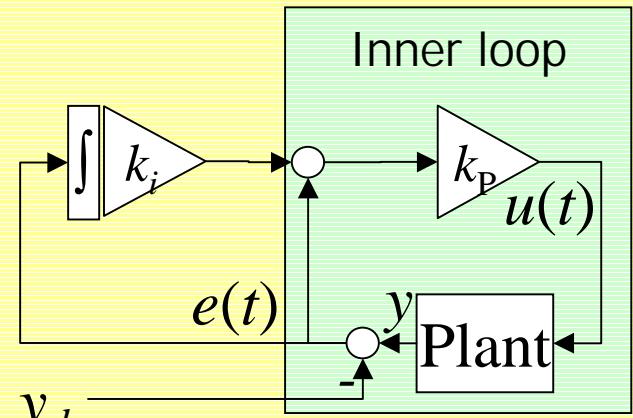
$$u = -k_I v - k_P e = k_P \left(e - \underbrace{k_i v}_{\frac{k_I}{k_P}} \right)$$

- Velocity form of the controller

$$\dot{u} = -k_I e - k_P \dot{e}$$

$$u(t+1) = u(t) - k_I e(t) - k_P [e(t) - e(t-1)]$$

Cascade loop interpretation:



$$k_I = k_i \cdot k_P$$

PI control

- Closed-loop dynamics

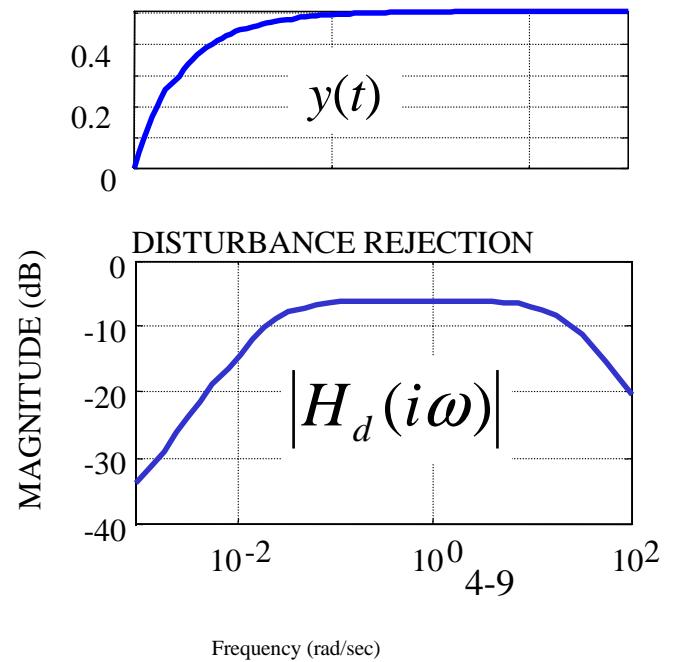
$$y = \frac{sk_P + k_I}{s(\tau s + 1) + sk_P + k_I} y_d + \frac{s}{s(\tau s + 1) + sk_P + k_I} d$$

- Steady state ($s = 0$): $y_{ss} = y_d$.
No steady-state error!
- Transient dynamics: look at the characteristic equation

$$\tau\lambda^2 + (1+k_P)\lambda + k_I = 0$$

- Disturbance rejection

$$|\hat{y}(i\omega)| = |H_d(i\omega)| \cdot |\hat{d}(i\omega)|$$



PLL Example

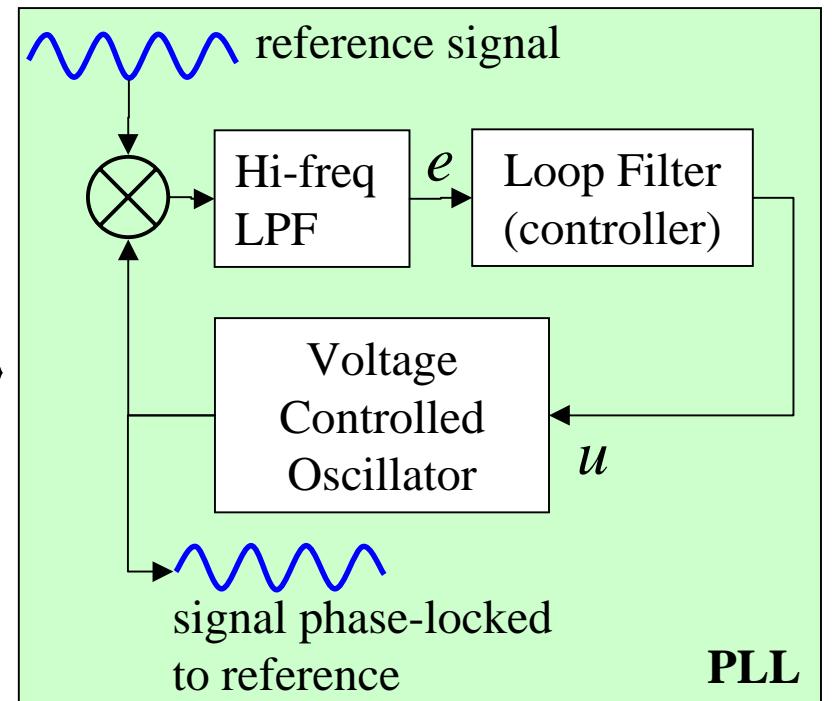
- Phase-locked loop is arguably a most prolific feedback system

$$e = 2K_m \text{LPF} \langle r \times v \rangle$$

$$= 2K_m \text{LPF} \langle A \sin(\omega t + \theta_d) \times \cos(\omega_o t + \theta_o) \rangle$$

$$e \approx AK_m \sin(\omega t - \omega_o t + \theta_d - \theta_o)$$

$$\dot{\theta}_o = \Delta\omega_o = K_o u$$



PLL Loop Model

- Small-signal model:

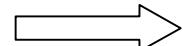
$$\theta = \omega t - \omega_o t + \theta_d - \theta_o \ll 1$$

$$e = K_d \sin(\theta) \approx K_d \theta$$

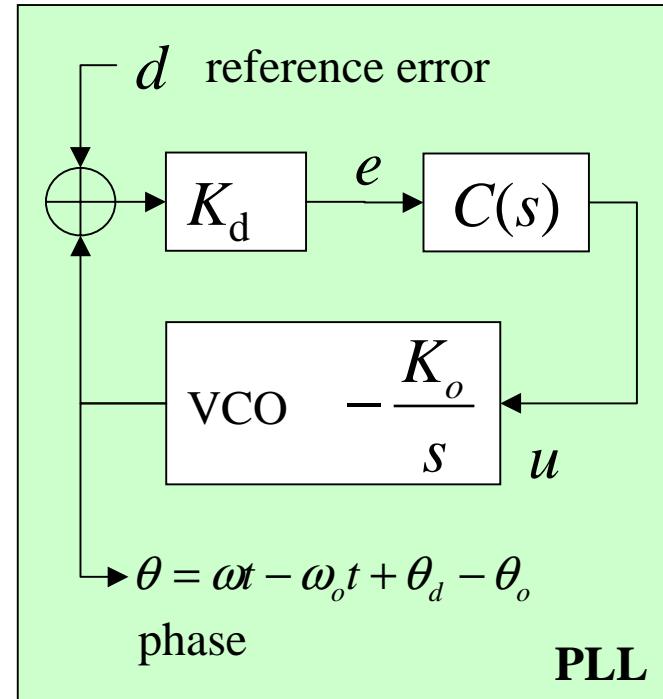
$$\dot{\theta} = \underbrace{\omega - \omega_o}_{d} + \underbrace{\dot{\theta}_d}_{\theta_o} - \frac{K_o u}{s}$$

- Loop dynamics:

$$\begin{aligned}\dot{\theta} &= d - K_o u \\ e &= K_d \theta \\ u &= k_P e + k_I \int e \cdot dt\end{aligned}$$



$$\theta = \frac{s}{s^2 + K_o K_d k_P s + k_I} d$$



PD control

- 2-nd order dynamics

$$\ddot{y} = u + d$$

- PD control

$$e = y - y_d$$

$$u = -k_D \dot{e} - k_P e$$

- Closed-loop dynamics

$$\ddot{e} + k_D \dot{e} + k_P e = d$$

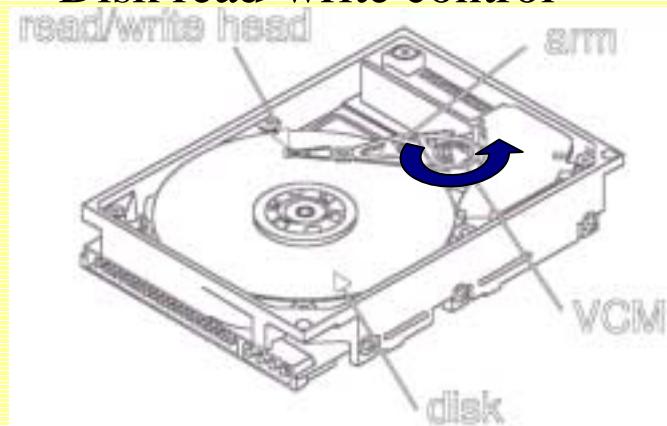
$$e = \frac{1}{s^2 + k_D s + k_P} d$$

- Optimal gains (critical damping)

$$k_D = 2\tau; \quad k_P = \tau^2$$

Example:

- Disk read-write control



$$J \ddot{\phi} = T_{VCM} + T_{DISTURB}$$



Voice
Coil
Motor

PD control

- Derivative (rate of e) can be obtained
 - speed sensor (tachometer)
 - low-level estimation logic
- Signal differentiation
 - is noncausal
 - amplifies high-frequency noise
- Causal (low-pass filtered) estimate of the derivative
$$\dot{e} \approx \frac{s}{\tau_D s + 1} e = \frac{1}{\tau_D} e + \frac{1/\tau_D}{\tau_D s + 1} e$$
- Modified PD controller:

$$u = -k_D \frac{s}{\tau_D s + 1} e - k_P e$$

PD control performance

- The performance seems to be infinitely improving for
 $k_D = 2\tau; k_P = \tau^2;$ $\tau \rightarrow \infty$
- This was a simple design model, remember?
- Performance is limited by
 - system being different from the model
 - flexible modes, friction, VCM inductance
 - sampling in a digital controller
 - rate estimation would amplify noise if too aggressive
 - actuator saturation
 - you might really find *after* you have tried to push the performance
- If high performance is really that important, careful application of more advanced control approaches might help

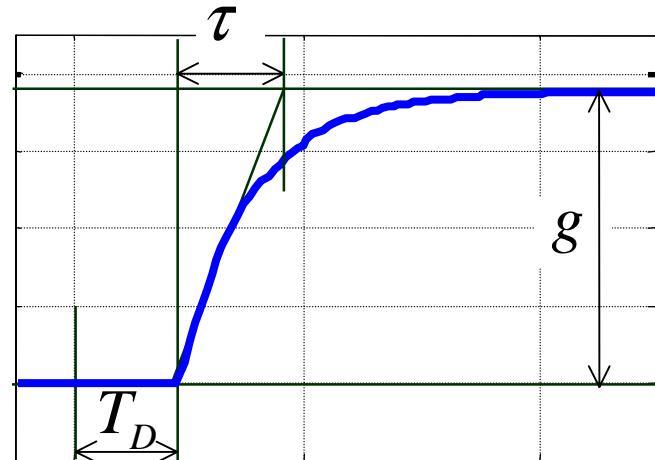
Plant Type

- Constant gain - I control
- Integrator - P control
- Double integrator - PD control
- Generic second order dynamics - PID control

PID Control

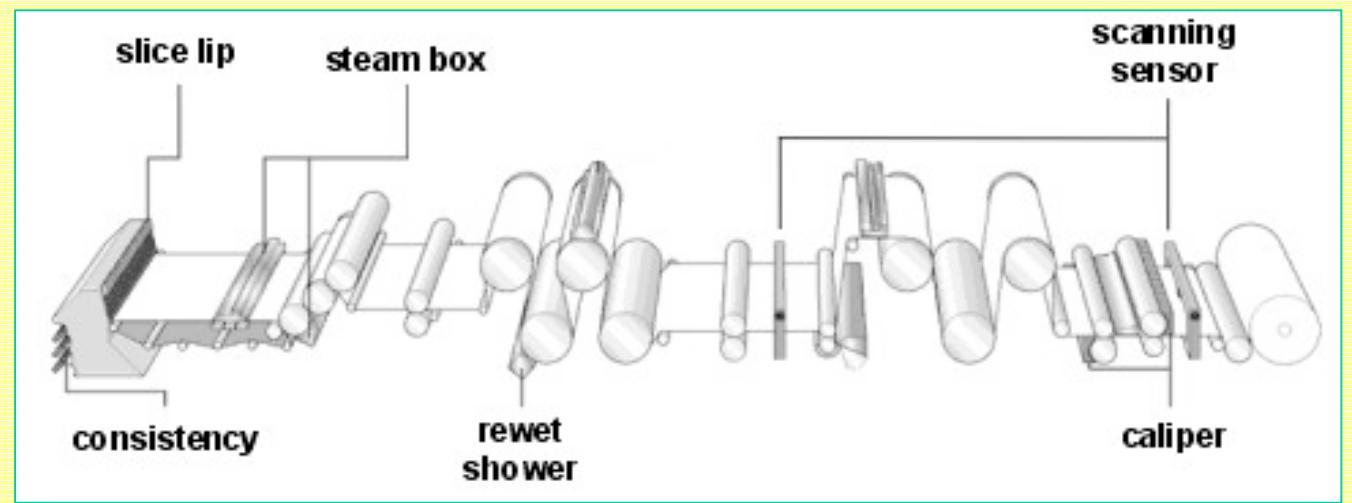
- Generalization of P, PI, PD
- Early motivation: control of first order processes with deadtime

$$y = \frac{ge^{-T_D s}}{\tau s + 1} u$$



Example:

- Paper machine control



PID Control

- PID: three-term control

$$e = y - y_d$$

*independent sensor
or an estimate*

$$u = -k_D \dot{e} - k_P e - k_I \int e \cdot dt$$

$$u = -k_D (1 - z^{-1}) e + k_P e + k_I \frac{1}{1 - z^{-1}} e$$

future

present

past

- Velocity form
 - bumpless transfer between manual and automatic

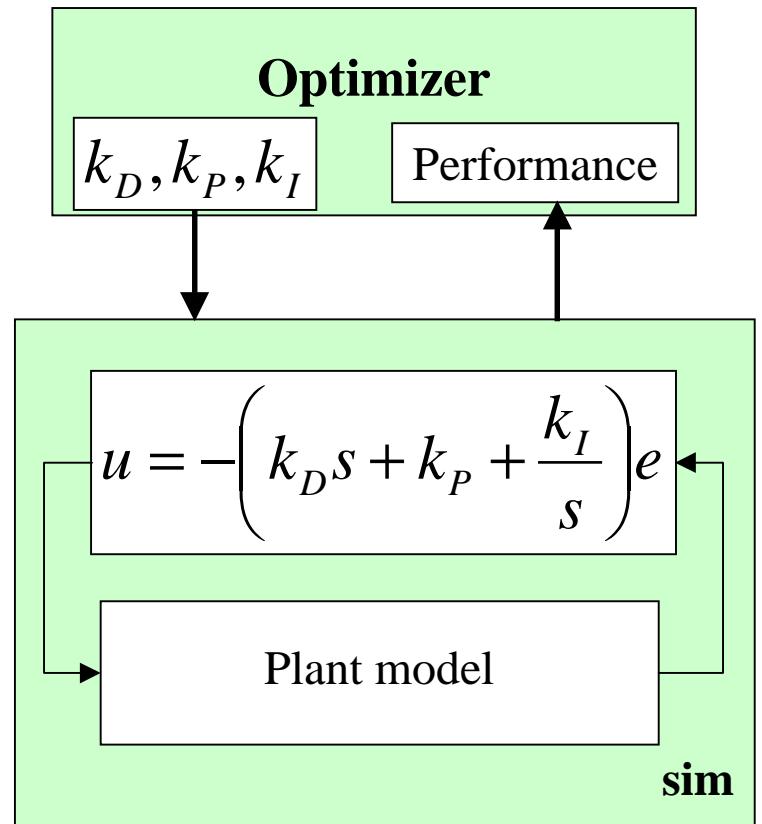
$$\Delta u = -k_D \Delta^2 e - k_P \Delta e - k_I e$$

$$\Delta = 1 - z^{-1}$$

$$u(t+1) = u(t) - k_I e(t) - k_P (e(t) - e(t-1)) \\ - k_D (e(t) - 2e(t-1) + e(t-2))$$

Tuning PID Control

- Model-based tuning
- Look at the closed-loop poles
- Numerical optimization
 - For given parameters run a sim, compute performance parameters and a performance index
 - Optimize the performance index over the three PID gains using grid search or Nelder method.



Zeigler-Nichols tuning rule

- Explore the plant:
 - set the plant under P control and start increasing the gain till the loop oscillates
 - note the critical gain k_C and oscillation period T_C

- Tune the controller:

	k_P	k_I	k_D
P	$0.5k_C$	—	—
PI	$0.45k_C$	$1.2k_P/T_C$	—
PID	$0.5k_C$	$2k_P/T_C$	$k_P T_C / 8$

- Z and N used a Monte Carlo method to develop the rule
- Z-N rule enables tuning if a model and a computer are both unavailable, only the controller and the plant are.

Integrator anti wind-up

- In practice, control authority is always limited:
 - $u_{MIN} \leq u(t) \leq u_{MAX}$
- Wind up of the integrator:
 - if $|u_c| > u_{MAX}$ the integral v will keep growing while the control is constant. This results in a heavy overshoot later
- Anti wind-up:
 - switch the integrator off if the control has saturated

$$\begin{aligned}\dot{v} &= e \\ u_c &= -k_I v - k_P e \\ u &= \begin{cases} u_{MAX}, & u_c > u_{MAX} \\ u_c, & u_{MIN} \leq u_c \leq u_{MAX} \\ u_{MIN}, & u_c < u_{MIN} \end{cases}\end{aligned}$$

$$\dot{v} = \begin{cases} e, & \text{for } u_{MIN} \leq u_c \leq u_{MAX} \\ 0, & \text{if } u_c > u_{MAX} \text{ or } u_c < u_{MIN} \end{cases}$$

Industrial PID Controller

- A box, not an algorithm
- Auto-tuning functionality:
 - pre-tune
 - self-tune
- Manual/cascade mode switch
- Bumpless transfer between different modes, setpoint ramp
- Loop alarms
- Networked or serial port

